

DESIGN MANUFACTURING AND ANALYSIS FOR OPTIMIZATION OF HEAT EXCHANGER BY USING FINS

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ABSTRACT

Heat exchangers play a vital role in thermal systems, and enhancing their heat has become increasingly important for improving energy efficiency. Conventional heat exchangers are often limited by insufficient surface area and low heat transfer coefficient limitations, pin fins small cylindrical or shaped projections attached to the base are used as extended surfaces to increase thermal performance. The present design, analysis, and performance evaluation of a heat exchanger integrated geometries. The incorporation of pin fins increases the effective surface area and which together enhance convective heat transfer. In this work, different conical, triangular, and square, materials (aluminium, copper, and brass), line and staggered arrangements) were examined. Among the tested configurations, staggered pin fin arrangements demonstrate enhancement due to better mixing of the working fluid. Optimized pin fin designs can improve the thermal effectiveness of the heat exchanger depending on geometry and flow conditions. This makes pin-fin-based a reliable, and cost-effective option for applications in automotive cooling plants, and electronic device thermal management. Crop yields under fluctuating climatic circumstances. Recently, researchers have used machine learning algorithms to forecast crop yields before actual planting. This research study has introduced a machine learning technique, namely linear regression and multilayer perceptron, to forecast crop production based on characteristics such as state, district, area, seasons, NPK, pH values, rainfall, temperature, and area.

KEYWORDS

Heat Exchanger (HE), Fin Design / Extended Surfaces, Thermal Management, Heat Transfer Enhancement, Thermal Performance, Thermal-Hydraulic Performance Computational Fluid Dynamics (CFD), Fin Efficiency / Fin Effectiveness, Fluid-Structure Interaction (FSI), Conjugate Heat Transfer (CHT), Material Selection, Micro-Manufacturing.

INTRODUCTION

Heat exchangers are fundamental thermal devices used in every major engineering industry to transfer heat efficiently from one fluid to another. Their importance has grown significantly due to global demand for energy efficiency, compact system design, and improved thermal performance in modern machines. Whether it is a refrigerator at home, a car radiator on the road, or a large heat exchanger in industrial processes, they ensure safe operating temperatures and optimal performance by controlling heat flow within the system. In conventional heat exchangers, the rate of heat transfer is often limited by the thermal resistance of the fluid boundary layer that forms over the heat-transfer surface. To overcome these limitations, engineers use extended surfaces known as fins, which increase the heat-transfer surface area and enhance convection. Among different types of fins—such as rectangular, triangular, and pin fins—the pin fin has emerged as one of the most effective designs for heat-transfer enhancement. Pin fins are small cylindrical or tapered protrusions

mounted on the base surface of a. When air or fluid flows over these pins, they break the boundary layer, create turbulence aid mixing. This results in a higher convective heat-transfer coefficient, allowing moved within the same volume. Pin fins also provide a three-dimensional heat-transfer efficient than traditional straight fins at exchangers using pin fins are particularly useful in applications where space, weight performance are critical factors. For example, modern electronics such as micro rules, and LED lighting systems generate significant heat in compact spaces. Pin-fin in safe temperatures without requiring bulky equipment. Similarly, in aerospace tries, pin fins are used for cooling turbine blades, exhaust systems, intercoolers, and se of their ability to provide high heat dissipation with minimal pressure drop. as are also easier to manufacture and can be arranged in various patterns such as to optimize the heat-transfer performance based on design requirements Mity (copper or steel) further expands their usage in different environment ons.t years, research has focused on improving pin-fin geometry by experiment in such as elliptical, square, hexagonal, and perforated pin fins. Such modification turbulence, improve cooling performance, and reduce aerodynamic drag.

PROBLEM STATEMENT

To design, analyze, and optimize a finned heat exchanger to maximize the overall heat transfer rate and minimize pumping power (pressure drop), while identifying the most cost-effective manufacturing processes and optimal fin geometries for a specific fluid application.

MOTIVATION

The motivation for optimizing finned heat exchangers lies in overcoming poor fluid thermal properties (like air-side resistance) and maximizing heat dissipation within strict size and weight constraints. This ensures greater energy efficiency, reduced manufacturing costs, and longer equipment lifespans across HVAC, aerospace, and power systems.

OBJECTIVES

- To Enhance the Heat-Transfer Rate
- To Enhance the Heat-Transfer Rate
- To Study the Effect of Fin Geometry
- To Evaluate Performance Under Different
- To Improve Energy Efficiency.

LITERATURE REVIEW

The development of heat exchangers using fins has evolved over more demanded higher efficiency and compact thermal systems, fins because technique in heat-transfer devices. The history can be understood through and technological progress.

Early Foundations (1800s-Early 1900s)

The concept of improving heat transfer by increasing surface area originate steam engines, boilers, and condensers required efficient heat regulation, transfer capability. Engineers soon realized that attaching extended surface increased the effective area and helped dissipate heat faster. Key developments during this period Use of metallic plates and strips around pipes to enhance cooling. Basic understanding of conduction and convection mechanisms. Development of early air-cooled heat exchangers in steam locomotive .These were not highly optimized fins but paved the way for finned heat .Birth of Fin Technology (1920s-1940s)

The industrial revolution and growth in manufacturing machinery better cooling solutions. Major advancements Introduction of mechanically attached fins on tubes to enhance heat. Use of circular fins (annular fins) in early air-cooled condensers an Development of copper and aluminium fins due to their high thermal Invention of the first forced convection finned heat exchangers in By the 1940s, finned-tube heat exchangers became common in auto plants. Post-War Expansion and Optimization (1950s 1970s)This period marked rapid growth due to advancements in thermodynamics, material

Key achievements:

- Introduction of precision-manufactured circular fins for engine cylinders (mot
- Development of plate-fin heat exchangers, tube-fin heat exchangers, and extent
- Use of mathematical models and analytical solutions to predict fin heat transfer
- Emergence of standardized fin geometries:

a) Circular Fins

b) Standard Fins

c) Square Fins

Circular fins became the preferred choice for cylindrical surfaces such as energy. heaters, and compact coolers due to excellent radial heat spread.

Modern Engineering and Computational Design (1980s-2000s)

With the introduction of computers and CFD (Computational Fluid Dynamics), scientific and optimized. Important developments Detailed modeling of fin performance under natural and forced convection. Use of simulation tools to optimize fin thickness, spacing, and material. Introduction of high-efficiency materials like alloys, composites, and coated Development of turbo-machinery fin systems for gas turbines and jet engines. Rapid expansion of electronic cooling systems, leading to heat sinks with Circular fins became very common in electronic heat sinks, refrigeration c exchangers. Contemporary Advances and

Innovations (2010s - Present)Recent years have seen highly optimized performance-driven fin designs d electronics, EVs, aerospace, and renewable energy systems. Modern advancements include Use of 3D printing to create complex circular fin geometries. Introduction of micro-fins and nano-fins for micro-electronics cooling.

COMPARISION

1. Design & Geometrical Analysis

- **Fin Profiles:** Louvered, offset strip (serrated), wavy, and pin-fins are the standard geometries evaluated in research.
- **Parameters Optimized:** Fin pitch, fin height, tube spacing, and angle of attack are primary design variables.
- **Optimization Methodologies:** Researchers typically use Computational Fluid Dynamics (CFD) coupled with metaheuristic algorithms—such as Genetic Algorithms (GA), Particle Swarm Optimization (PSO), or Artificial Neural Networks (ANN)—to identify optimal geometric dimensions.

2. Manufacturing & Material Selection

- **Material Constraints:** Aluminium and copper remain the industry standards due to their high thermal conductivity.
- **Manufacturing Techniques:**
 - **Traditional:** Mechanically expanded tubes or brazed fin-and-tube assemblies.
 - **Advanced:** Additive manufacturing (3D printing) enables the creation of complex, topology-optimized fin arrays (e.g., lattice or biomimetic structures) that methods cannot achieve.
- **Trade-off:** High-conductivity materials offer better heat transfer but may have higher structural costs and manufacturing complexity.

3. Thermal-Hydraulic Performance Analysis

- **Evaluation Metrics:** The objective is to maximize the heat transfer coefficient and the Colburn factor, while minimizing pumping power and the Fanning friction factor
- **Performance Index:** Effectiveness-NTU method and Thermal Enhancement Factor (TEF) are the standard indicators

METHODOLOGY

1. Conceptual Design & Mathematical Modeling

- **Define Parameters:** Establish the boundary conditions (mass flow rates, inlet temperatures, fluid properties) and operating constraints.
- **Governing Equations:** Select fin configurations (e.g., plate, louvered, or pin fins) and use classical sizing methods, specifically the **Logarithmic Mean Temperature Difference (LMTD)** method or the **Effectiveness-Number of Transfer Units (NTU)** method.

2. Numerical Analysis & Optimization

- **Computational Fluid Dynamics (CFD):** Build 3D CAD models and mesh the geometry. Use simulation software like **Ansys** to simulate fluid flow and thermal behavior by solving Navier-Stokes and energy equations.
- **Performance Indicators:** Evaluate the Colburn

-factor (heat transfer) and Fanning

-factor (pressure drop).

- **Multi-Objective Optimization:** Optimize geometric variables—such as fin pitch, thickness, height, and tube layout. Implement optimization algorithms like **Genetic Algorithm (GA)**, **Particle Swarm Optimization (PSO)**, or **Response Surface Methodology (RSM)** to find the Pareto optimal design.

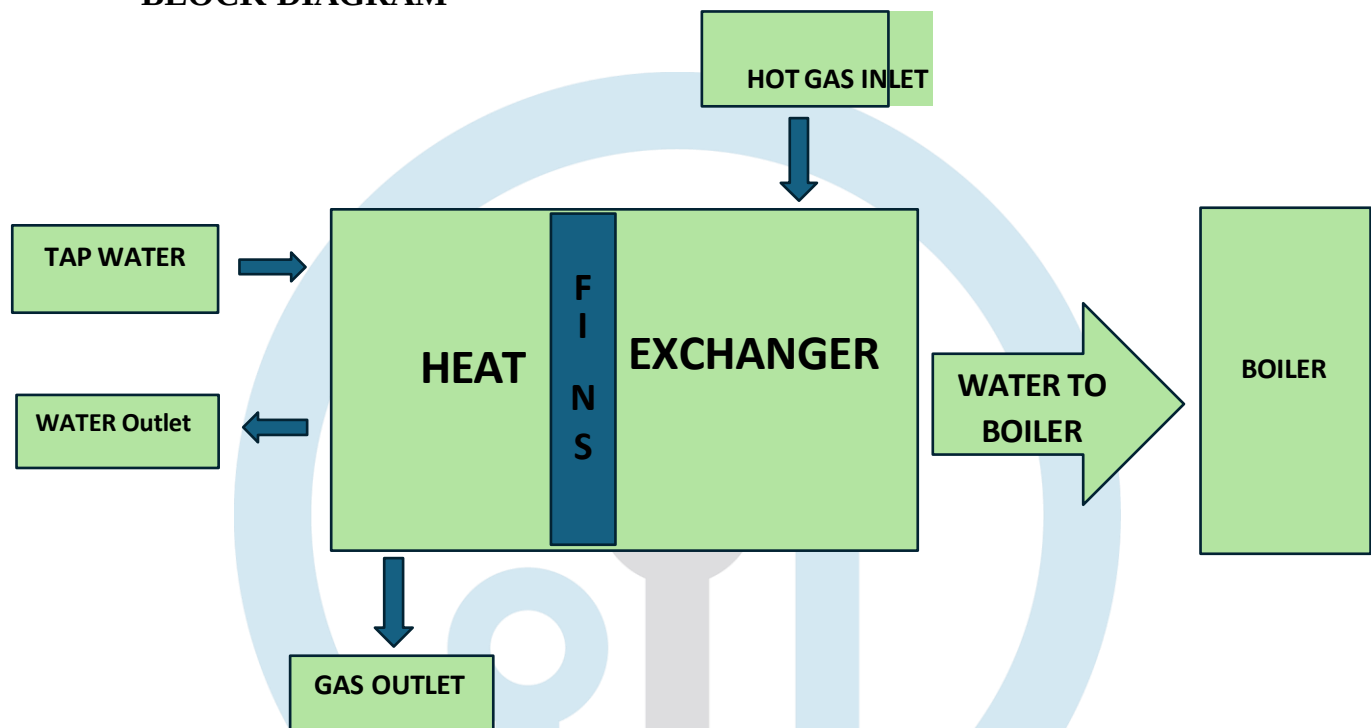
3. Manufacturing Processes

- **Material Selection:** Choose materials based on thermal conductivity and operating environment (e.g., copper, aluminium, or stainless steel).
- **Fabrication Techniques:**
 - Mechanical bonding: Expanding tubes into collars of plate fins.
 - Brazing: Using controlled atmosphere brazing (CAB) to bond fins to tubes for automotive/HVAC applications.
 - Additive Manufacturing (3D Printing): Increasingly used for producing complex, topology-optimized, single-unit fin structures.

4. Experimental Testing & Validation

- **Testing Rig:** Construct an experimental loop to measure inlet/outlet temperatures, mass flow rates, and pressure drops across the manufactured prototype.
- **Validation:** Compare experimental data with your analytical and CFD models. Adjust empirical correlations and evaluate the thermo-hydraulic performance to confirm the final optimization is verified for research paper publication

BLOCK DIAGRAM



CONCEPTUAL DESIGN

1. CASING



Fig. Casing

2. TUBE



Fig. Tube

3. BUFFEL PLATE



Fig. Baffle Plate

4. INLET HEDER



Fig. Inlet Header

5. OUTLET HEDER



Fig. Outlet Header

6. SAEDDEL

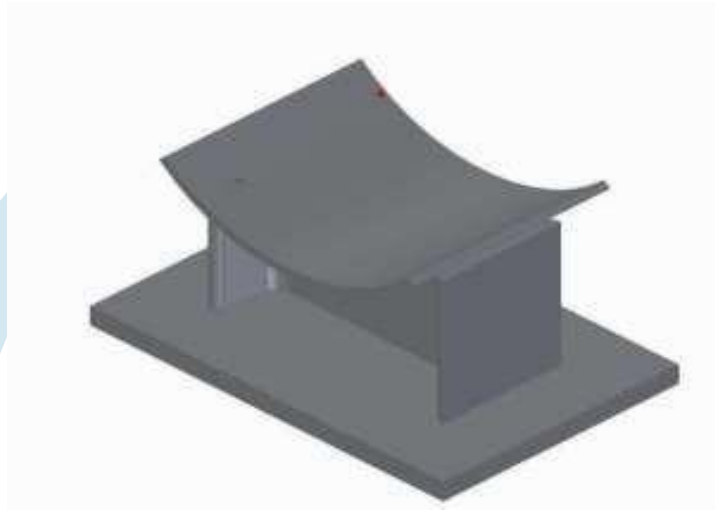


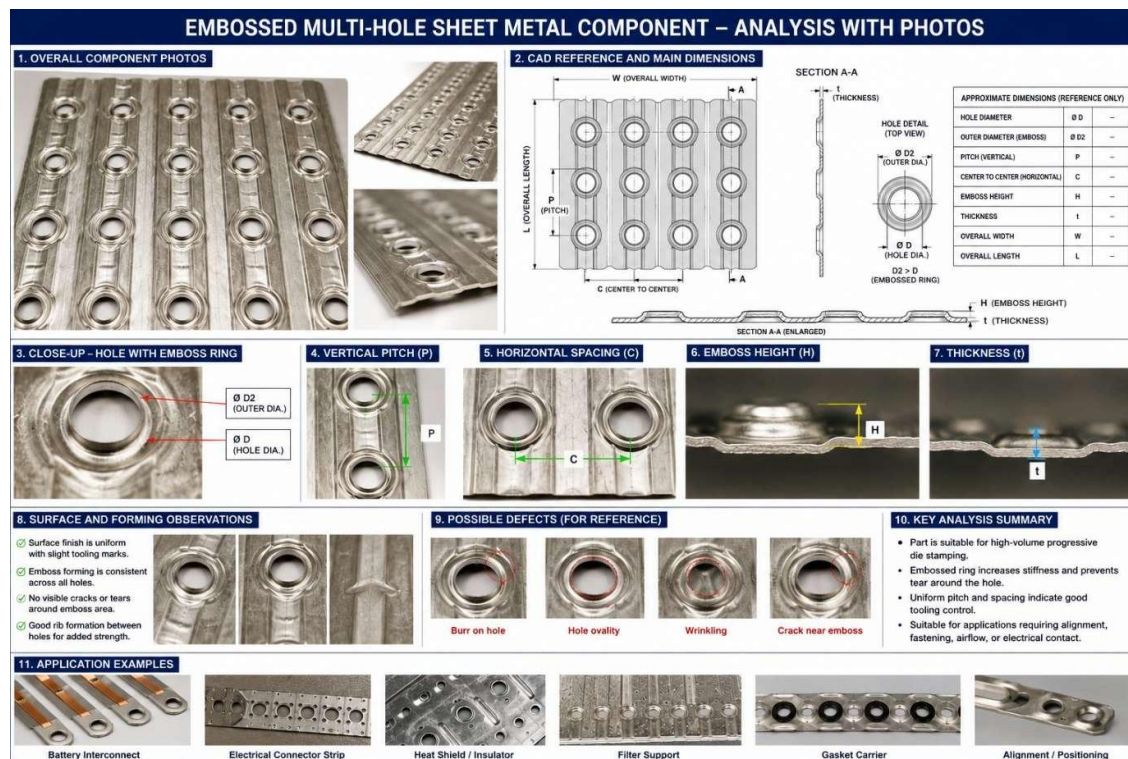
Fig. Saddle

7. Assembly Final Product



Fig. After Manufacturing And Assembly Final Product

8. FINS



CALCULATIONS

Hot Fluid (Water)

- Inlet temperature of hot fluid:

$$T_{h,in} = 90^{\circ}\text{C}$$

- Outlet temperature of hot fluid:

$$T_{h,out} = 60^{\circ}\text{C}$$

Cold Fluid (Air)

- Inlet temperature:

$$T_{c,in} = 30^{\circ}\text{C}$$

- Outlet temperature:

$$T_{c,out} = 50^{\circ}\text{C}$$

Other Data

- Mass flow rate:

$$m = 0.5 \text{ kg/s}$$

- Specific heat of water:

$$C_p = 4180 \text{ J/kgK}$$

- Overall heat transfer coefficient:

$$U = 120 \text{ W/m}^2\text{K}$$

- Fin Material: Aluminium

$$k = 205 \text{ W/mK}$$

- Heat transfer coefficient:

$$h = 60 \text{ W/m}^2\text{K}$$

- Fin dimensions:

- Length:

$$L = 0.05 \text{ m}$$

- Width:

$$W = 0.10 \text{ m}$$

- Thickness:

$$t = 0.002 \text{ m}$$

1. HEAT TRANSFER RATE

Formula:

$$Q = mC_p(T_{out} - T_{in})$$

Substitute values:

$$Q = 0.5 \times 4180 \times (90 - 60)$$

$$Q = 0.5 \times 4180 \times 30$$

$$Q = 62700 \text{ W}$$

Answer:

$$Q = 62.7 \text{ kW}$$

2. LOG MEAN TEMPERATURE DIFFERENCE (LMTD)

Step 1: Temperature Differences

$$\Delta T_1 = T_{h,in} - T_{c,out}$$

$$\Delta T_1 = 90 - 50 = 40^\circ\text{C}$$

$$\Delta T_2 = T_{h,out} - T_{c,in}$$

$$\Delta T_2 = 60 - 30 = 30^\circ\text{C}$$

Step 2: LMTD Formula

$$LMTD = \frac{\Delta T_1 - \Delta T_2}{\ln (\Delta T_1 / \Delta T_2)}$$

$$LMTD = \frac{40 - 30}{\ln (40/30)}$$

$$LMTD = \frac{10}{\ln (1.333)}$$

$$LMTD = 0.2877$$

$$LMTD = 34.75^\circ\text{C}$$

Answer:

$$LMTD = 34.75^\circ\text{C}$$

3. REQUIRED HEAT TRANSFER AREA

Formula:

$$Q = UA(LMTD)$$

Rearranging:

$$A = \frac{Q}{ULMTD}$$

Substitute values:

$$A = \frac{62700}{120 \times 34.75}$$

$$A = \frac{62700}{4170}$$

$$A = 15.03 \text{ m}^2$$

Answer:

$$A = 15.03 \text{ m}^2$$

4. FIN SURFACE AREA

Formula:

$$A_f = 2LW$$

$$A_f = 2(0.05)(0.10)$$

$$A_f = 0.01 \text{ m}^2$$

Answer:

$$A_f = 0.01 \text{ m}^2$$

5. FIN PERIMETER

Formula:

$$P = 2(W + t)$$

$$P = 2(0.10 + 0.002)$$

$$P = 0.204 \text{ m}$$

Answer:

$$P = 0.204 \text{ m}$$

6. CROSS-SECTIONAL AREA OF FIN

Formula:

$$A_c = W \times t$$

$$A_c = 0.10 \times 0.002$$

$$A_c = 0.0002 \, m^2$$

Answer:

$$A_c = 2 \times 10^{-4} \, m^2$$

7. FIN PARAMETER (m)

Formula:

$$m = \sqrt{\frac{hP}{kA_c}}$$

Substitute values:

$$m = \sqrt{\frac{60 \times 0.204}{205 \times 0.0002}}$$

$$m = \sqrt{\frac{12.24}{0.041}}$$

$$m = \sqrt{298.53}$$

$$m = 17.27$$

Answer:

$$m = 17.27$$

8. FIN EFFICIENCY

Formula:

$$\eta_f = \frac{\tanh (mL)}{mL}$$

Calculate:

$$mL = 17.27 \times 0.05$$

$$mL = 0.8635$$

$$\tanh(0.8635) = 0.698$$

$$\eta_f = \frac{0.698}{0.8635}$$

$$\eta_f = 0.808$$

Answer:

$$\eta_f = 80.8\%$$

9. HEAT TRANSFER THROUGH FIN

Formula:

$$Q_f = \sqrt{hPkA_c}(T_b - T_\infty)\tanh(mL)$$

Assume:

- Base temperature:

$$T_b = 90^\circ\text{C}$$

- Surrounding air temperature:

$$T_\infty = 30^\circ\text{C}$$

Temperature difference:

$$(T_b - T_\infty) = 60^\circ\text{C}$$

Step 1

$$\begin{aligned} &\sqrt{hPkA_c} \\ &= \sqrt{60 \times 0.204 \times 205 \times 0.0002} \\ &= \sqrt{0.50184} \\ &= 0.708 \end{aligned}$$

Step 2

$$\begin{aligned} Q_f &= 0.708 \times 60 \times 0.698 \\ Q_f &= 29.64 \text{ W} \end{aligned}$$

Answer:

$$Q_f = 29.64 \text{ W}$$

10. REYNOLDS NUMBER

Assume:

- Density:

$$\rho = 1.2 \text{ kg/m}^3$$

- Velocity:

$$V = 5 \text{ m/s}$$

- Hydraulic diameter:

$$D_h = 0.02 \text{ m}$$

- Dynamic viscosity:

$$\mu = 1.8 \times 10^{-5}$$

Formula:

$$Re = \frac{\rho V D_h}{\mu}$$

$$Re = \frac{1.2 \times 5 \times 0.02}{1.8 \times 10^{-5}}$$

$$Re = \frac{0.12}{1.8 \times 10^{-5}}$$

$$Re = 6666$$

Answer:

$$Re = 6666$$

(Turbulent Flow)

11. NUSSELT NUMBER

Formula:

$$Nu = 0.023 Re^{0.8} Pr^{0.4}$$

Assume:

$$Pr = 0.71$$

$$Nu = 0.023(6666)^{0.8}(0.71)^{0.4}$$

$$Nu = 0.023(1147)(0.872)$$

$$Nu = 23.0$$

Answer:

$$Nu = 23$$

12. CONVECTIVE HEAT TRANSFER COEFFICIENT

Formula:

$$h = \frac{Nu \times k}{D_h}$$

Air conductivity:

$$k = 0.026 \text{ W/mK}$$

$$23 \times 0.026$$

$$h = 0.02$$

$$h = 29.9 \text{ W/m}^2\text{K}$$

Answer:

$$h = 29.9 \text{ W/m}^2\text{K}$$

13. PRESSURE DROP

Assume friction factor:

$$f = 0.03$$

Length:

$$L = 1\text{m}$$

Formula:

$$\Delta P = f \frac{L}{D_h} \frac{\rho V^2}{2}$$

$$\Delta P = 0.03 \times \frac{1}{0.02} \times \frac{1.2 \times 5^2}{2}$$

$$\Delta P = 0.03 \times 50 \times 15$$

$$\Delta P = 22.5 \text{ Pa}$$

Answer:

$$\Delta P = 22.5 \text{ Pa}$$

FUTURE SCOPE

The future scope of heat exchanger optimization using fins is very wide due to the increasing demand for efficient thermal systems in industries such as power plants, automobiles, refrigeration, air conditioning, chemical processing, aerospace, and renewable energy systems. The use of fins in heat exchangers improves the heat transfer rate by increasing the surface area available for heat exchange. Future developments in this field mainly focus on improving thermal efficiency, reducing energy consumption, minimizing size and weight, and increasing the overall performance of heat exchangers.

CONCLUSION

The future scope of fin-based heat exchanger optimization is very broad due to increasing demand for:

- Higher thermal efficiency
- Compact systems
- Energy conservation
- Smart thermal management

Advanced materials, AI-based optimization, nano-fluids, and modern manufacturing technologies will play a major role in the future development of high-performance heat exchangers.

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